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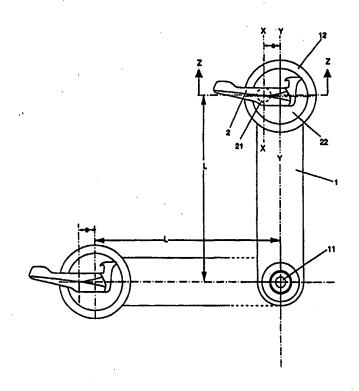
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| 71)(72) Applicant and Inventor: BATTISTUTTA, [IT/IT]; Via Torsa, 23, I-33050 Ariis di Rivigna | Antoi ino (IT). | io |
| 74) Agent: D'AGOSTINI, Giovanni; Via G. Giusti, 1 Udine (IȚ). | 7, I–331 | 00 |
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(54) Title: PEDAL CRANK DEVICE FOR A BICYCLE AND BICYCLE USING THE SAME

(57) Abstract

Pedal-pedal crank advancement control device for a bicycle, of the type in which there are two opposite, equal and symmetrical pedal-pedal cranks, one on one side and one on the other side of the bicycle frame, engaged to it and keyed with said engagement to a gear wheel intended for driving the motion by a chain to a freewheel axially keyed to the back wheel of the same bicycle, where each pedal crank (1) involves a pedal (2) engaged to its free end, characterized in that it has a system for keying the pedal to the pedal crank with respect to its substantial engagement, independently from the pedal crank angular position.



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WO 99/41140 PCT/IT98/00141

| 1 | DESCRIPTION |
|----|---|
| 2 | |
| 3 | PEDAL CRANK DEVICE FOR A BICYCLE AND BICYCLE USING THE SAME |
| 4 | |
| 5 | Technical Field |
| 6 | The present invention has for object a pedal-pedal crank advancement |
| 7 | control device for bicycle and respective bicycle with said pedal-pedal |
| 8 | crank. |
| 9 | Background Art |
| 10 | In prior art the traditional pedal-pedal crank control devices for |
| 11 | bicycle advancement are known, in which the pedal engages to the |
| 12 | pedal crank and the pedal crank engages in its turn to the bicycle |
| 13 | frame where by means of a gear wheel it drives the motion by chain to |
| 14 | the back wheel of the same bicycle. |
| 15 | For obviating certain speed conditions and torque need on the pedal the |
| 16 | traditional bicycles are traditionally provided with a gear-change |
| 17 | which acts both on the pedal crank gear wheel and on the sprocket |
| 18 | (generally a freewheel one) coaxial to the back wheel (generally on |
| 19 | both by means of different gearing diameters). |
| 20 | The prior art drawbacks substantially derive from that the engagement |
| 21 | points between frame pedal crank and between pedal crank and pedal |
| 22 | are fixed thus some upper dead point and lower dead point conditions |
| 23 | (high pedal on one side and low pedal on the other side) are created |
| 24 | which make the push exerted by the foot in said conditions little |
| 25 | effective. |
| 26 | A further problem derives from that the pedal and pedal crank |
| 27 | structures are always critical and always need a suitable dimensioning |
| 28 | for avoiding the risk of breakage. |

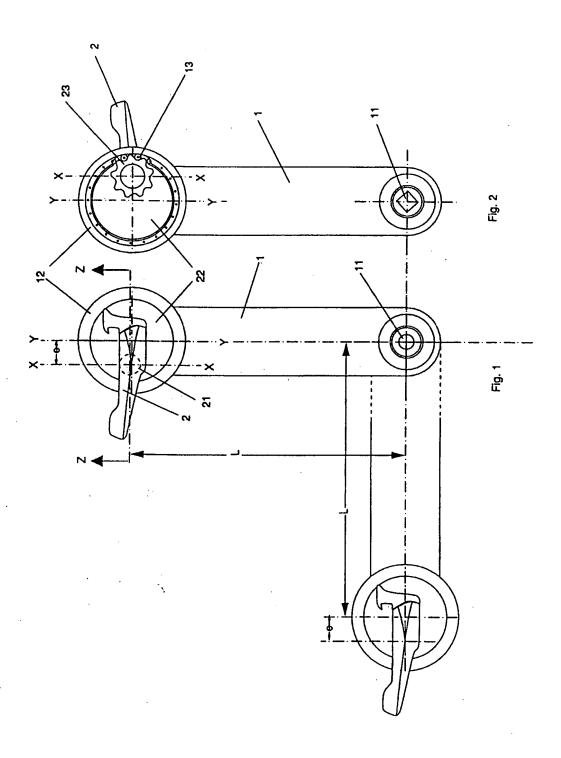
- 1 Another drawback derives from the very high concentration of the
- 2 pushes which subject some bending stresses to concentration
- 3 particularly in the engagement zones, compromising the effectiveness
- 4 and the life of the respective bearings.
- 5 Furthermore the dimensioning involves an increase of the bicycle
- 6 weight to the detriment of the advancement yield.
- 7 Purpose of the present invention is that of obviating the above
- 8 mentioned drawbacks.
- 9 The problem is solved as claimed by means of a pedal-pedal crank
- 10 advancement control device for bicycle and respective bicycle with
- said pedal-pedal crank, of the type in which there are two opposite
- 12 alike and symmetrical pedals-pedal crank, one on one side and one on
- 13 the other side of the bicycle frame, engaged to it and keyed by said
- engagement to a gear wheel intended to drive the motion by a chain to
- a freewheel axially keyed to the back wheel of the same bicycle, where
- each pedal crank has engaged to its free end a pedal,
- 17 characterized in that:
- 18 said pedal is rigidly fastened in forward eccentric position respect to
- 19 the advancement direction when the pedal is substantially positioned at
- 20 the upper dead point, to a disk-shaped flange substantially axially
- 21 engaged along the middle of said pedal crank;
- 22 on the eccentric keying axis of said pedal, a gear sprocket is
- 23 rotationally free engaged which is internally keyed to a crown gear
- 24 which is integrated into the containing crown of said disk-shaped
- 25 flange in said free end of said pedal crank;
- 26 being the gearing pitch such that during said pedal crank rotation
- 27 the eccentric keying axis of said pedal, is always placed forward
- 28 substantially parallel to the resting plane of said pedal.

- 1 Thus an extremely advantageous structure is obtained, mainly
- 2 functional and reliable, able to give a bigger middle torque and thus
- 3 also a higher yield.
- 4 Thus the keying system of the pedal to the pedal crank keeps the pedal
- 5 keying axis always placed forward respect to its substantial
- 6 engagement, independently from the pedal crank angular position,
- 7 thus making the pedal crank arm (distance from the centre of the pedal
- 8 to the pedal crank rotation centre) variable according to the pedal
- 9 crank angular position.
- 10 Advantageously the pedal-pedal crank control device for bicycle is
- 11 further characterized in that said rotationally idle gear sprocket is
- 12 gearing in said pedal crank crown gear on the internal side of the same
- that is on the opposite side of said pedal keying disk-shaped flange.
- 14 Thus an optimal structure is obtained.
- 15 These and other advantages will appear from the following description
- 16 of a preferential embodiment solution, with the aid of the enclosed
- 17 drawings, whose execution details are not to be considered as limitative
- 18 but only given as examples.
- 19 Figure 1 is a view of the pedal-pedal crank substantially in two vertical
- and horizontal positions showing the forward constant position of the
- 21 pedal with varied distance during the pedal crank rotation (greater
- 22 when it is horizontal).
- 23 Figure 2 is a back view of the pedal-pedal crank in vertical position as
- 24 in Fig. 1, showing the sprocket-crown gearing of the pedal to the pedal
- 25 crank.
- 26 Fig. 3 is a Z-Z axial section view respect to Fig. 1 of the pedal-pedal
- 27 crank gearing.
- According to the figure it can be noticed that:

- 1 The pedal crank is indicated with 1 and the pedal with 2.
- 2 The pedal crank has a holed end for the engagement to the bicycle
- 3 frame (11) for the keying of the respective driving gear wheel to the
- 4 chain, while the other opposite free end has an housing crown 12
- 5 which on one side houses a disk-shaped flange (22) rotationally free to
- 6 rotate, rigidly keyed by eccentric axis with value "e" to the pedal (2).
- 7 On this eccentric axis (X-Y; 22) a rotationally free gear sprocket (23)
- 8 which gears in an internal crown gear (13) of the pedal crank (1) end,
- 9 is keyed on the internal part.
- 10 The sprocket-crown (23-13) gearing is obtained so that to a pedal crank
- 11 turn corresponds an inverse turn of the pedal, so that keeping the
- 12 pedal always horizontal, independently on the pedal crank (1) angular
- position respect to its rotation centre (11), the eccentricity "e" of the
- keying axis (22) of the pedal (2) is always placed forward, substantially
- 15 obtaining:
- when the pedal is high, it is placed forward respect to the rotation axis
- vertical of the pedal crank (11) of a value "e" while the pedal distance
- from the pedal crank rotation centre is equal to a value "L" and a motor
- moment equal to "P"x"e" is obtained, where "e" is the eccentricity and
- 20 "P" is the vertical force exerted on the pedal.
- 21 when the pedal is all placed forward (horizontal pedal crank) the
- 22 pedal distance from the pedal crank rotation centre is progressively
- 23 increased to a value (L+"e") increasing the respective moment and
- 24 therefore the torque,
- 25 vice versa for the opposite positions of the other pedal.
- Thus a continuous variability during the pedal stroke of the effective
- pedal stroke arm defined by the pedal (2) distance from the pedal crank
- 28 (11) rotation centre is obtained.

- 1 The structure realized further allows a higher drive strength and a
- decidedly better division of the forces on the wider rotational surfaces
- 3 respect to the present engagement systems, with consequent
- 4 mechanism longer life and resistance and performance.
- 5 The rather frequent engagement breakage in the pedal is thus also
- 6 avoided.
- 7 The pedal forced by the foot to remain always horizontal, forces the
- 8 effective pedal crank arm, thanks to the eccentricity and constrained
- 9 sprocket-crown (23-13) gearing, to vary continuously according to an
- 10 optimal condition.

| 1 | Claims |
|------|---|
| 2 | 1. Pedal-pedal crank advancement control device for bicycle, of the |
| 3 | type in which there are two opposite, equal and symmetrical pedals- |
| 4 | pedal cranks, one on one side and one on the other side of the bicycle |
| 5 | frame, engaged to it and keyed with said engagement to a gear wheel |
| 6 | intended for driving the motion by a chain to a freewheel axially keyed |
| 7 | to the back wheel of the same bicycle, where each pedal-crank (1) |
| 8 | involves a pedal (2) engaged to its free end, characterized in that: |
| 9 | - said pedal (2) is rigidly fastened in forward eccentric position (X-X 21) |
| 0 | respect to the advancement direction when the pedal (2) is |
| 1 | substantially positioned at the upper dead point, to a disk-shaped flange |
| .2 | substantially axially engaged (Y-Y) along the middle of said pedal |
| .3 | crank (1); |
| .4 | - on the eccentric keying axis (21) of said pedal (2), a gear sprocket is |
| .5 | rotationally free engaged which is internally keyed to a crown gear |
| 6 | (13) which is integrated into the containing crown of said disk-shaped |
| 7 | flange in said free end of said pedal crank (1); |
| 8 | - being the gearing pitch such that during said pedal crank (1) rotation |
| 9 | the eccentric keying axis (21) of said pedal (2), is always placed forward |
| 20 | substantially parallel to the resting plane (Z-Z) of said pedal (2). |
| 21 | 2. Pedal-pedal crank advancement control device for bicycle according |
| 22 | to claim 1, characterized in that said rotationally idle gear sprocket (21) |
| 23 . | is gearing in said pedal crank crown gear (12) on the internal side of |
| 24 | the same that is on the opposite side of said pedal (2) keying disk |
| 25 | shaped flange (22). |
| 26 | 3. Bicycle provided with a couple of pedals-pedal crank according to |
| 27 | any of the previous claims. |



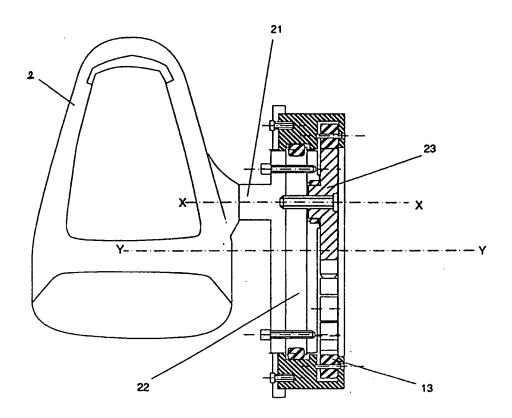


Fig. 3 sec. Z-Z

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